

# CS-184: Computer Graphics

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## Lecture #22: Rigid Body Dynamics

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V2006-F-22-1.0

## Today

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- Rigid-body dynamics
- Articulated systems

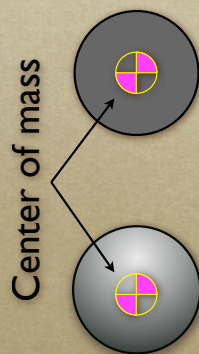
# A Rigid Body

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- A solid object that does not deform
  - Consists of infinite number of infinitesimal mass points...
  - ...that share a single RB transformation
    - Rotation + Translation (no shear or scale)
$$\mathbf{x}^W = \mathbf{R} \cdot \mathbf{x}^L + \mathbf{t}$$
    - Rotation and translation vary over time
  - Limit of deformable object as  $k_S \rightarrow \infty$

# A Rigid Body

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In 2D:

Translation 2 “directions”

Rotation 1 “direction”

3 DOF Total

In 3D:

Translation 3 “directions”

Rotation 3 “direction”

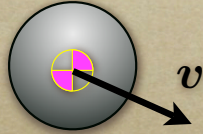
6 DOF Total

Translation and rotation are *decoupled*

2D is boring... we'll stick to 3D from now on...

# Translational Motion

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Just like a point mass:

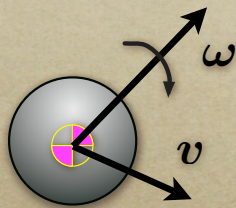
$$\dot{\mathbf{p}} = \mathbf{v}$$

$$\dot{\mathbf{v}} = \mathbf{a} = \mathbf{f}/m$$

Note: Recall discussion on integration...

# Rotational Motion

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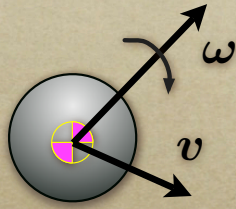


Rotation gets a bit odd, as well see...

Rotational "position"  $R$   
Rotation matrix  
Exponential map  
Quaternions

Rotational velocity  $\omega$   
Stored as a vector  
(Also called angular velocity...)  
Measured in radians / second

# Rotational Motion



Kinetic energy due to rotation:

“Sum energy (from rotation) over all points in the object”

$$E = \int_{\Omega} \frac{1}{2} \rho \dot{\mathbf{x}} \cdot \dot{\mathbf{x}} \, du$$

$$E = \int_{\Omega} \frac{1}{2} \rho ([\omega \times] \mathbf{x}) \cdot ([\omega \times] \mathbf{x}) \, du$$

# Rotational Motion

$H = \frac{\partial E}{\partial \omega}$  H momentum (angular)  
(work/rot)  
 $H_p = \frac{\partial E}{\partial \omega_p}$   
 $= \int_{\Omega} \rho \frac{1}{2} (\epsilon_{ijk} \delta_{jp} x_k \epsilon_{iab} \omega_a x_b + \epsilon_{ijk} \omega_j x_k \epsilon_{ipa} \delta_{pa} x_b) \, du$   
 $= \int_{\Omega} \rho \frac{1}{2} (\epsilon_{ijk} x_k \epsilon_{iab} \omega_a x_b + \epsilon_{ijk} \omega_j x_k \epsilon_{ipa} x_b) \, du$   
 $= \int_{\Omega} \rho \epsilon_{ijk} x_k \epsilon_{iab} \omega_a x_b \, du$   
 $= \int_{\Omega} \rho \mathbf{x} \times (\omega \times \mathbf{x}) \, du$   
 $= \int_{\Omega} \rho \mathbf{x} \times \mathbf{v} \, du$   
 $= \omega_a \int_{\Omega} \rho \epsilon_{ijk} x_k \epsilon_{iab} x_b \, du$   
 $= \omega_a \int_{\Omega} \rho (\delta_{pa} \delta_{kb} - \delta_{pb} \delta_{ka}) x_k x_b \, du$   
 $= \omega_a \int_{\Omega} \rho (\delta_{pa} x_k x_k - x_a x_p) \, du$   
 \*  $H_p = I_{ap} \omega_a$  Inertia Tensor

angular momentum to linear momentum derived from rotational energy

$$\mathbf{H} = \int_{\Omega} \rho \mathbf{x} \times \dot{\mathbf{x}} \, du$$

$$\mathbf{H} = \int_{\Omega} \rho \mathbf{x} \times (\omega \times \mathbf{x}) \, du$$

$$\mathbf{H} = \left( \int_{\Omega} \dots \, du \right) \omega$$

“Inertia Tensor” not identity matrix...

$$\mathbf{H} = \mathbf{I} \omega$$

# Inertia Tensor

$$\mathbf{I} = \int_{\Omega} \rho \begin{bmatrix} y^2 + z^2 & -xy & -xz \\ -xy & z^2 + x^2 & -yz \\ -xz & -yz & x^2 + y^2 \end{bmatrix} du$$

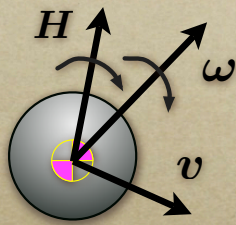
See example for simple shapes at

<http://scienceworld.wolfram.com/physics/MomentofInertia.html>

Can also be computed from polygon models by transforming volume integral to a surface one.

See paper/code by Brian Mirtich.

# Rotational Motion



Conservation of momentum:

$$\mathbf{H}^W = \mathbf{I}^W \boldsymbol{\omega}^W$$

$$\mathbf{H}^W = \mathbf{R} \mathbf{I}^L \mathbf{R}^T \boldsymbol{\omega}^W$$

Figure is a lie if this really is a sphere...

$$\dot{\mathbf{H}}^W = \dot{\mathbf{R}} \mathbf{I}^L \mathbf{R}^T \boldsymbol{\omega}^W + \mathbf{R} \mathbf{I}^L \dot{\mathbf{R}}^T \boldsymbol{\omega}^W + \mathbf{R} \mathbf{I}^L \mathbf{R}^T \boldsymbol{\alpha}^W$$

$$\dot{\mathbf{H}}^W = 0$$

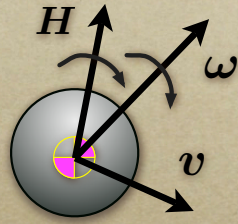
$$\dot{\mathbf{R}} = \boldsymbol{\omega} \times \mathbf{R}$$

$$\boldsymbol{\alpha}^W = (\mathbf{R} \mathbf{I}^L \mathbf{R}^T)^{-1} (-\boldsymbol{\omega}^W \times \mathbf{H}^W)$$

In other words, things wobble when they rotate.

# Rotational Motion

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$$\dot{R} = [\omega \times] R$$

$$\dot{\omega} = \alpha$$

Figure is a lie if this really is a sphere...

$$\alpha^W = (RI^L R^T)^{-1} ((-\omega^W \times H^W) + \tau)$$

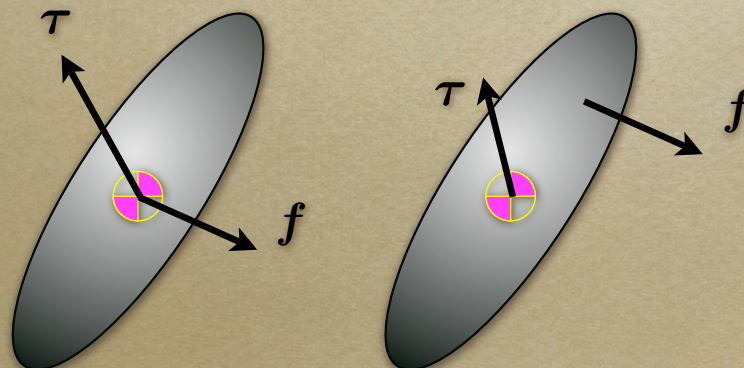
$$\tau = f \times x$$

Take care when integrating rotations, they need to stay rotations.

# Couples

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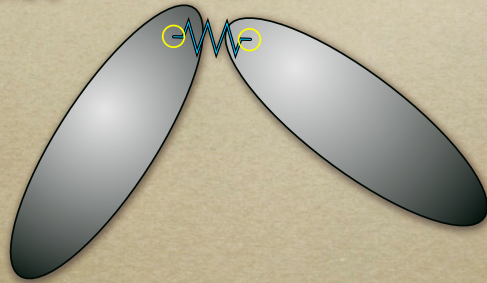
- A force / torque pair is a couple
  - Also a wrench (I think)
- Many couples are equivalent



# Constraints

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- Simplest method is to use spring attachments
  - Basically a penalty method



- Spring strength required to get good results may be unreasonably high
  - There are ways to cheat in some contexts...

# Constraints

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- **Articulation constraints**
  - Spring trick is an example of a full coordinate method
    - Better constraint methods exist
  - Reduced coordinate methods use DOFs in kinematic skeleton for simulation
    - Much more complex to explain
- **Collisions**
  - Penalty methods can also be used for collisions
  - Again, better constraint methods exist

# Suggested Reading

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• Brian Mirtich, "Fast and Accurate Computation of Polyhedral Mass Properties," Journal of Graphics Tools, volume 1, number 2, 1996. <http://www.cs.berkeley.edu/~jfc/mirtich/papers/vollnt.ps>

• Brian Mirtich and John Canny, "Impulse-based Simulation of Rigid Bodies," in Proceedings of 1995 Symposium on Interactive 3D Graphics, April 1995. <http://www.cs.berkeley.edu/~jfc/mirtich/papers/ibsrp.ps>

• D. Baraff, Linear-time dynamics using Lagrange multipliers. Computer Graphics Proceedings, Annual Conference Series: 137-146, 1996. <http://www.pixar.com/companyinfo/research/deb/sig96.pdf>

• D. Baraff, Fast contact force computation for nonpenetrating rigid bodies. Computer Graphics Proceedings, Annual Conference Series: 23-34, 1994. <http://www.pixar.com/companyinfo/research/deb/sig94.pdf>